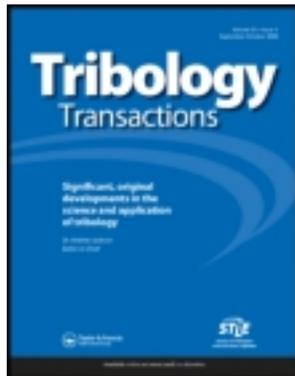


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Tribology Transactions

Publication details, including instructions for authors and subscription information:

<http://www.tandfonline.com/loi/utrb20>

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R. González^a, A. Hernández Battez^b, J. L. Viesca^b, A. Higuera-Garrido^b & A. Fernández-González^c

^a University of Oviedo, Department of Marine Science and Technology Gijón, Asturias, Spain

^b University of Oviedo, Department of Construction and Manufacturing Gijón, Asturias, Spain

^c University of Oviedo, Department of Physical and Analytical Chemistry Asturias, Spain

Accepted author version posted online: 19 Jun 2013. Published online: 06 Aug 2013.

To cite this article: R. González, A. Hernández Battez, J. L. Viesca, A. Higuera-Garrido & A. Fernández-González (2013) Lubrication of DLC Coatings with Two Tris(pentafluoroethyl)trifluorophosphate Anion-Based Ionic Liquids, Tribology Transactions, 56:5, 887-895, DOI: [10.1080/10402004.2013.810319](https://doi.org/10.1080/10402004.2013.810319)

To link to this article: <http://dx.doi.org/10.1080/10402004.2013.810319>

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Lubrication of DLC Coatings with Two Tris(pentafluoroethyl)trifluorophosphate Anion-Based Ionic Liquids

R. GONZÁLEZ,¹ A. HERNÁNDEZ BATTEZ,² J. L. VIESCA,² A. HIGUERA-GARRIDO,²
and A. FERNÁNDEZ-GONZÁLEZ³

¹University of Oviedo

Department of Marine Science and Technology
Gijón, Asturias, Spain

²University of Oviedo

Department of Construction and Manufacturing
Gijón, Asturias, Spain

³University of Oviedo

Department of Physical and Analytical Chemistry
Asturias, Spain

The lubrication of a Cr-DLC coating with ethyl-dimethyl-2-methoxyethylammonium tris(pentafluoroethyl)trifluorophosphate [(NEMM)MOE][FAP] and 1-butyl-1-methylpyrrolidinium tris(pentafluoroethyl)trifluorophosphate [BMP][FAP] ionic liquids (ILs) as 1 wt% additives to a polyalphaolefin (PAO 6) was studied. Zinc dialkyldithiophosphate (ZDDP) was also used as reference in order to evaluate the effectiveness of the ILs. Reciprocating ball-on-plate tribological tests at loads of 20 and 40 N were performed. The results showed that both ILs exhibited a friction reduction, especially at the lowest load tested. Antiwear properties were also improved; the PAO 6 + 1% [BMP][FAP] mixture was slightly better, close to the values for PAO 6 + 1% ZDDP. Scanning electron microscopy (SEM) images and X-ray photoelectron spectroscopy (XPS) analysis indicated that the additive-surface interaction was responsible for the tribological improvement.

KEY WORDS

DLC; Ionic Liquid; Lubrication; Additive

INTRODUCTION

Since their discovery, diamond-like carbon (DLC) films have become one of the most valuable engineering materials for a number of industrial applications, including microelectronics, transportation, manufacturing, and biomedical fields (Aisenberg and Chabot (1)). They simultaneously exhibit improved wear protection, high hardness, excellent corrosion resistance, high

thermal and chemical stability, and a low coefficient of friction (Hauert (2); Semenov and Khrushchov (3); Scharf and Singer (4); Voevodin, et al. (5); Erdemir and Fenske (6)). Due to all of these excellent tribological properties, the popularity of DLC coatings during the last decade has increased in engineering applications such as engine tribocomponents to improve material durability and fuel economy (Haque, et al. (7)).

DLC coatings are becoming quite widely used for machine elements in lubricated systems, especially in the automotive industry (He, et al. (8); Gangopadhyay, et al. (9)), as well as under high stress and/or under boundary or mixed lubrication regimes (Kalin, et al. (10); Manier, et al. (11)). Topolovec, et al. (12) studied the friction properties of a range of boundary lubricating additives in DLC rolling-sliding contacts and analyzed how these coatings interacted with the various additives present in the formulated lubricants and the impact of additive-DLC interactions on friction and wear. Kalin, et al. (10) studied the DLC-lubricant interaction and analyzed only DLC-DLC contacts in order to avoid the inevitable effects from interactions with other counter materials such as steel. Barros, et al. (13) described the friction and wear properties of DLC coatings in boundary lubrication conditions as a function of the hydrogen content in the carbon coating, concluding that hydrogenated DLC coatings can be better lubricated in the presence of the friction modifier additive molybdenum dialkyldithiocarbamate (MoDTC) through the formation of an MoS₂ solid lubricant. These works summarized the present understanding of the boundary lubrication of DLC coatings. Miyake, et al. (14) showed that the microwear resistance and nanoindentation hardness can be improved by adding materials like titanium (Ti), molybdenum (Mo), and iron (Fe) to DLC films under lubricated contacts with both MoDTC and zinc dialkyldithiophosphate (ZDDP) additives. Haque, et al. (7) addressed the friction and wear performance of a nonhydrogenated DLC coating as a function of additive chemistry and showed

that the antiwear additive ZDDP provided excellent wear protection by forming zinc phosphate in the tribofilm. However, when a moly dimer (MD) or moly trimer (MT) friction modifier was added with ZDDP, zinc phosphate was found to be absent, resulting in higher wear than ZDDP alone.

On the other hand, due to the strong interaction of their components through the Coulomb force, ionic liquids (ILs), unlike normal liquids that are bonded through van der Waals forces (Kamimura, et al. (15)), have remarkable physical and chemical properties, such as inherent polarity, higher thermal stability, nonflammability, and low volatility (Yu, et al. (16)), that make them a greener alternative to conventional organic solvents and potential lubricants (Palacio and Bhushan (17); Minami (18); Bermúdez, et al. (19)). One of the most interesting features of these new lubricants is their use under extreme conditions for which conventional lubricants fail (Bermúdez, et al. (19); Yao, et al. (20); Street, et al. (21); Jiménez and Bermúdez (22); Morales, et al. (23)). In summary, ILs show better tribological behavior than commonly used synthetic, mineral, or engine oils on different lubricated surfaces. Moreover, tribology efforts in this field must focus on the surface interactions between the material and the ionic liquid, which defines the efficiency of the ionic liquid as a lubricant (Minami (18); Zhang, et al. (24); Qu, et al. (25), (26); Song, et al. (27)).

Early works in the tribological applications of ionic liquids was focused on a variety of BF_4 and PF_6 imidazolium salts (as additives or neat lubricant) that showed good results with regard to both antifriction and antiwear performance (Yu, et al. (16); Yao, et al. (20); Arora and Cann (28); Battez, et al. (29); Viesca, et al. (30)). However, the mentioned imidazolium salts are hydrophilic, and some problems related to the formation of hydrogen fluoride and subsequent corrosive damage in tribological systems have been reported (Minami, et al. (31)). In recent years, new ILs based on fluorinated structures, such as bis(trifluoromethylsulfonyl)imide [NTf_2] and tris(pentafluoroethyl)trifluorophosphate [FAP] have been developed (Pisarova, et al. (32)). Both the high hydrophobicity and hydrolytic stability of the [FAP] anion allowed overcoming the main drawbacks of the traditional BF_4 and PF_6 anions (Minami, et al. (31)).

Although there are many studies on the use of ionic liquids as lubricants on different types of engineering surfaces such as steel (Yao, et al. (20); Battez, et al. (29)), aluminum (Qu, et al. (25); Somers, et al. (33)), titanium (Jiménez and Bermúdez (22), (34)), and nickel alloys (Jiménez, et al. (35)), only a few papers have been published using DLC coatings with the presence of ionic liquids as neat lubricants or additives. Sasaki (36) combined the advantages of ILs and DLC coatings to develop a novel tribosystem for high-vacuum sliding applications. Gonzalez, et al. (37) studied 1-butyl-1-methylpyrrolidinium tris(pentafluoroethyl)trifluorophosphate IL as neat lubricant and 1 wt% additive to a polyalphaolefin (PAO 6) in the lubrication of physical vapor deposition (PVD) coatings at high contact pressure (mean contact pressure of 1.54 and maximum contact pressure of 2.32 GPa). Trying to extend the service life of mild steel components by applying combined surface modification of nitriding and DLC coating, Jia, et al. (38), (39) focused on the friction and wear behavior of DLC coatings on mild steel substrate under boundary lubricated conditions using synthetic IL functionalized borate esters and benzotriazole-containing borate esters as additives in a polyalphaolefin.

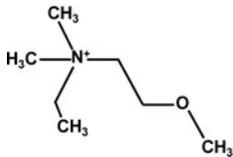
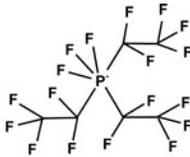
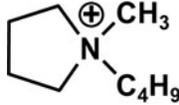
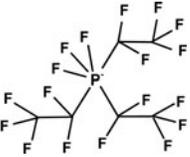
Considering the huge increase of DLC industrial applications in the presence of lubrication and the continuous development of novel ILs with potential use in tribology, this paper studies the friction and wear behavior of Cr-DLC PVD coating lubricated with two tris(pentafluoroethyl)trifluorophosphate anion-based ILs used as an additive to PAO 6 and compared this behavior to that of a traditional oil additive such as ZDDP.

EXPERIMENTAL DETAILS

Table 1 shows the characteristics of the two ILs used as oil additives: ethyl-dimethyl-2-methoxyethylammonium tris(pentafluoroethyl)trifluorophosphate [(NEMM)MOE][FAP] and 1-butyl-1-methylpyrrolidinium tris(pentafluoroethyl)trifluorophosphate [BMP][FAP].

A polyalphaolefin (PAO 6) with a density of 0.826 g/cm^3 at 15°C (ASTM D 4052), a kinematic viscosity of 31.0 and $5.90 \text{ mm}^2/\text{s}$ at 40 and 100°C , respectively (ASTM D 445), and a viscosity index of 135 (ASTM D 2270) was employed as base oil.

TABLE 1—Ionic Liquids Properties

Ionic Liquid	Cation	Anion	Purity (%)	Density (g/mL)	Water Content
[(NEMM)MOE][FAP]			99	1.78	<100 ppm (Karl Fisher)
[BMP][FAP]			99	1.59	<1%

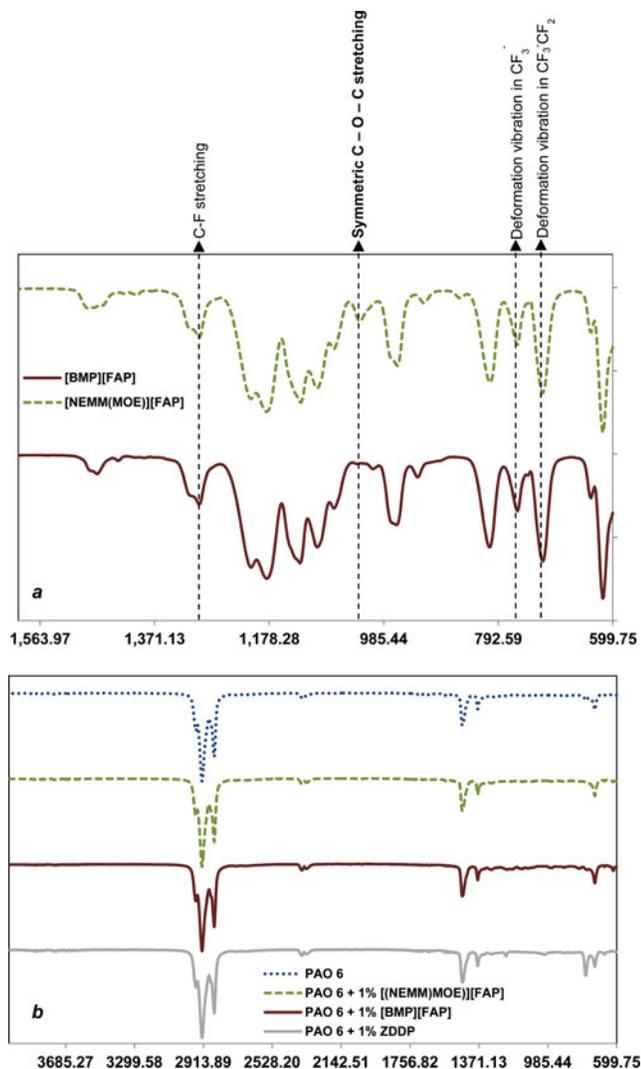


Fig. 1—ATR-FTIR spectrum of (a) pure ionic liquids and (b) base oil and all of the mixtures studied (color figure available online).

A traditional oil additive, ZDDP, was used as a reference in order to measure the effectiveness of the ILs as additives. In particular, a phosphorodithioic acid, mixed O,O-bis(iso-Bu and pentyl) esters, zinc salts, was employed for this purpose.

Attenuated total reflection-Fourier transform infrared spectroscopy (ATR-FTIR) was used to examine pure [(NEMM)MOE][FAP] and [BMP][FAP] as well as the four lubricant samples (pure base oil, two base oil-IL mixtures, and a base oil-ZDDP mixture). Both ILs had very similar ATR-FTIR spectra due to the common fluorinated anion FAP^- (Fig. 1a). Both showed peaks at 717 cm^{-1} (C-F deformation vibration in $CF_3^-CF_2^-$), 760 cm^{-1} (C-F deformation vibration in CF_3^-), and 1295 cm^{-1} (C-F stretching) assignable to fluorinated carbon chains. Additionally, bands at 808 and 902 cm^{-1} are expected to appear in compounds with F-P bonds. The main difference lies in the peak at 1025 cm^{-1} (symmetric C-O-C stretching) appearing in the [(NEMM)MOE][FAP] IL due to the existence of an ether bond that is not present in [BMP][FAP]. Dilutions of 1 wt% of the ILs in PAO 6 showed spectra very close to that of the

neat base oil because of the low concentration (Fig. 1b). PAO 6 showed two intense bands at 2850 and 2918 cm^{-1} belonging to C-H stretching vibrations of the aliphatic carbon chain and a medium intensity band at 1460 cm^{-1} assignable to the C-H deformation of the aliphatic $-CH_3$.

An ultrasonic probe was employed to prepare the base oil-additive mixtures. Each of the additives (ILs and ZDDP) was added to the PAO 6 at a concentration of 1.0 wt%. A point to be considered in the use of ILs as additives is the stability of the blends. The difference in polarity with the base oil can cause problems related to possible immiscibility. In order to guarantee correct dispersion of the ILs in the base oil during tribological tests (30 min), mixtures were studied by means of a multiple light scattering analyzer. Results of the backscattering of light as a function of time showed that both blends remained practically constant for 5 h, and the PAO 6 + 1% [(NEMM)MOE][FAP] was the most stable. However, further studies on the stability of the ILs are needed. On the other hand, the ZDDP additive was completely soluble in the base oil.

A ball-on-plate reciprocating configuration was chosen to evaluate the tribological behavior of the different lubricants. Cr-DLC PVD coatings (thickness: $3.2\text{ }\mu\text{m}$, hardness: 700 HV) were deposited over ASTM A-569 steel plates ($10 \times 10 \times 3\text{ mm}^3$). AISI E52100 chrome steel balls (diameter: 9.5 mm, hardness: 63 RC) were used as a counterbody. Friction and wear tests with the above configuration were performed in a UMT-3 microtribometer. Room temperature tests of 30 min duration in the presence of 15 mL of the corresponding lubricant sample, at 1 mm of amplitude, 10 Hz of reciprocating frequency (corresponding with a mean sliding speed of 0.02 m/s), and two different normal loads of 20 and 40 N (corresponding to mean contact pressures of 0.9 and 1.14 GPa, respectively) were performed. Three tests were conducted for each condition and lubricant sample, and friction force and normal load were monitored online.

After the tribological tests, the wear volume of the coated plates was quantified using a white light interferometer (Zygo New View 5000). Three-dimensional images were treated as follows: The tilt from the original surface was removed by subtracting a plane (surface leveling). This plane was calculated using a plane fitting algorithm based on least squares regression. The inputs of the algorithm were the unworn zones of the original surface. These areas were identified and carefully selected as polynomial regions on the original surface. After leveling the original surface, the wear scar was selected. The wear volume was numerically evaluated by solving the corresponding integral of volume with Simpson's rule. The surface morphology of the wear tracks was studied by a scanning electron microscopy with energy dispersive spectroscopy (SEM-EDS). X-ray photoelectron spectrometry (XPS) analysis was carried out in order to understand the interaction between the lubricants and the coated surfaces.

RESULTS AND DISCUSSION

Friction

Figure 2 shows the evolution of the friction coefficient during tests performed at 20 N for the four lubricant samples studied. Results for pure PAO 6 showed the highest value due to

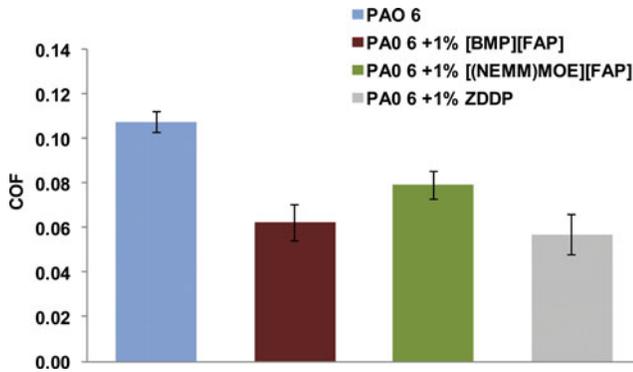


Fig. 2—Friction coefficient evolution during tests performed at 20 N (color figure available online).

the absence of friction reduction additives. Both ILs used as additives reduced the friction compared to the base oil. However, tests lubricated with PAO 6 + 1% [BMP][FAP] exhibited lower coefficient of friction values than those obtained for PAO 6 + 1% [(NEMM)MOE][FAP] and values close to those obtained for PAO 6 + 1% ZDDP, which presented the lowest friction results.

The expected increasing trend of the friction coefficient for each lubricant tested when the applied load increased can be seen in a comparison of the results obtained at 40 N (Fig. 3) with those obtained at 20 N (Fig. 2). Once again, the use of ILs as additives improved the anti-friction properties of the base oil. Nevertheless, the friction reduction obtained during tests with the mixtures in comparison with the base oil was lower at 40 N than that recorded at 20 N; the mixture PAO 6 + 1% [BMP][FAP] was slightly better than that of PAO 6 + 1% [(NEMM)MOE][FAP], as in the tests at a 20 N load. On the other hand, the mixture PAO 6 + 1% ZDDP showed similar behavior under both loads and had the lowest friction coefficient. The highest friction coefficient was obtained for PAO 6 and the anti-friction performance of all additives used (ILs and ZDDP) were improved under both loads.

Table 2 shows the reduction in the friction coefficient achieved for each mixture with regard to the base oil under the two loads used. As can be observed, the use of both ILs led to a great reduction in the friction coefficient at 20 N, reaching almost 42% in

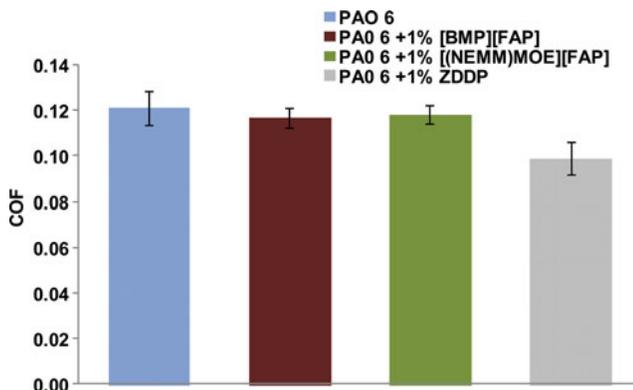


Fig. 3—Average friction coefficient values for tests performed at 40 N (color figure available online).

TABLE 2—FRICTION COEFFICIENT REDUCTION COMPARE TO VALUES FOR PURE PAO 6 FOR BOTH LOADS

Lubricant	Reduction in the Coefficient of Friction (%)	
	20 N	40 N
PAO 6 + 1% [(NEMM)MOE][FAP]	26.16	2.47
PAO 6 + 1% [BMP][FAP]	41.90	4.13
PAO 6 + 1% ZDDP	47.02	19.01

the case of PAO 6 + 1% [BMP][FAP] and exceeding 26% for the PAO 6 + 1% [(NEMM)MOE][FAP] mixture. However, ZDDP exhibited superior behavior as a friction reduction additive. On the other hand, a decrease in friction for all mixtures at 40 N was also observed, although ILs only resulted in only a slight reduction compared to those for pure PAO 6. However, ZDDP showed excellent behavior as an additive.

After the friction tests, the wear volume from the coated plates was measured. Except for pure PAO 6, no measurable wear was detected from surfaces tested at 20 N. Figure 4 shows SEM images of wear tracks lubricated with the base oil and the three mixtures. As can be seen in Figs. 4b and 4c, both ILs significantly reduced the surface damage compared to the base oil (Fig. 4a). According to the friction tests, ZDDP exhibited the best performance and the wear track was almost negligible when the PAO 6 + 1% ZDDP blend was used as the lubricant (Fig. 4d).

In Fig. 5 it can be observed that significant wear tracks were found after tests made at 40 N. Three-dimensional images show damage on the coated surfaces lubricated with pure PAO 6 (Fig. 5a). The depth of the wear scar and the absence in this area of the coating elements found later in the EDS analysis confirmed the total destruction of the Cr-DLC layer during the test made with the base oil. On the other hand, smaller wear tracks were found for each of the three additives studied, indicating better antiwear performance of the mixtures compared to that of the pure PAO 6 (Figs. 5b–5d).

Figure 6 shows the wear volume measured after tests at the highest load. It can be observed that an important wear reduction resulted when the two ILs were used as additives. However, PAO 6 + 1% [BMP][FAP] exhibited better performance than PAO 6 + 1% [(NEMM)MOE][FAP], with results slightly worse than those obtained for PAO 6 + 1% ZDDP.

SEM-EDS analysis confirmed the tribological results for tests performed at 40 N. No coating elements were found on the tracks of the surfaces lubricated with pure PAO 6, but a large presence of iron from the substrate was detected. Nevertheless, some Cr appeared on the worn surfaces tested with the three additives, revealing that the Cr-DLC layer was not totally removed.

These results were in accordance with previous authors' works (Blanco, et al. (40), (41)) that studied the tribological behavior of CrN and TiN PVD coatings lubricated with [(NEMM)MOE][FAP] IL as 1 wt% additive to PAO 6. In these works, IL-PAO 6 mixtures exhibited an important friction reduction compared to pure base oil; however, ZDDP blends showed the best performance of the two PVD coatings studied. A

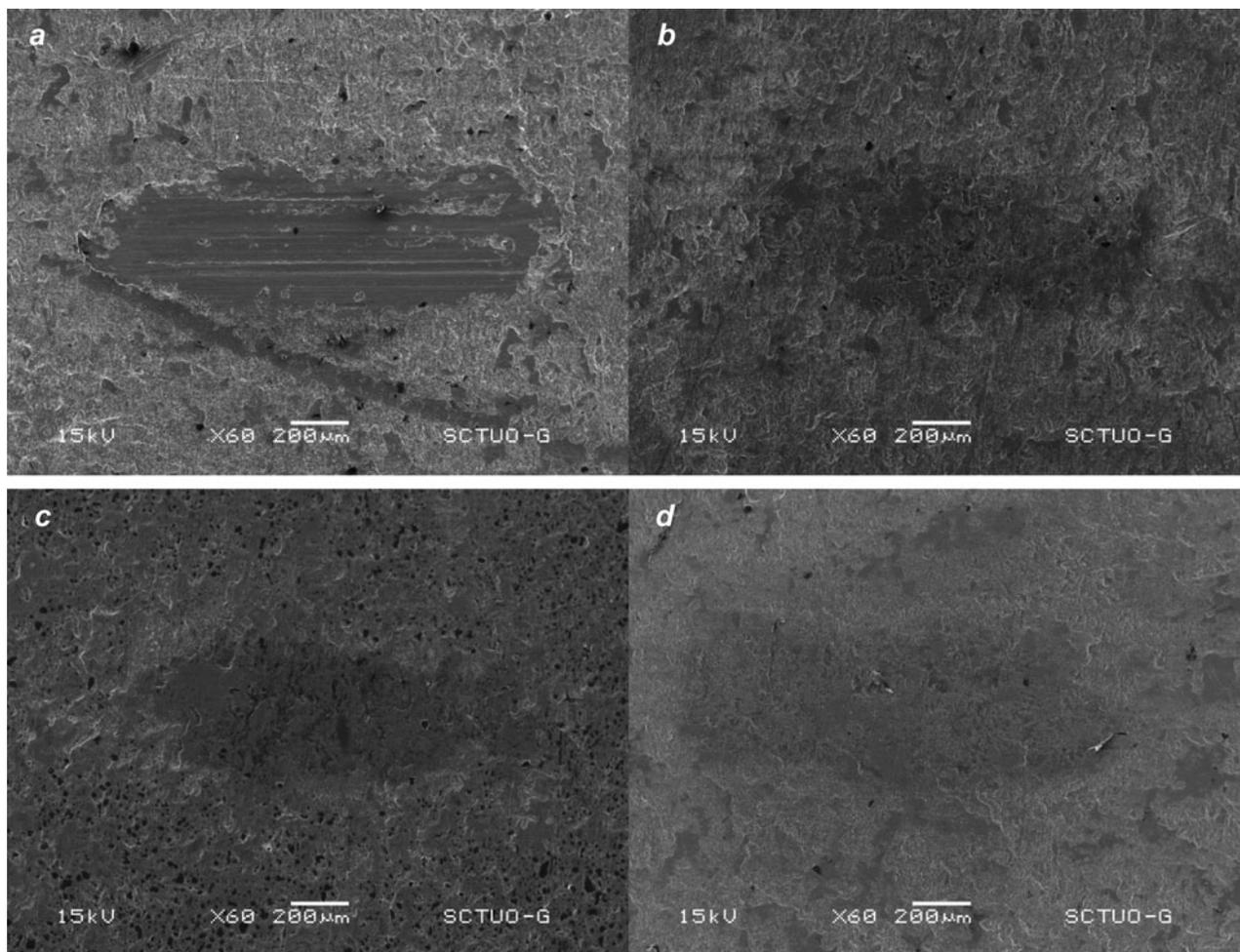


Fig. 4—SEM images of wear tracks on the DLC coating after tribological tests at 20 N lubricated with (a) PAO 6, (b) PAO 6 + 1% [BMP][FAP], (c) PAO 6 + 1% [(NEMM)MOE][FAP], and (d) PAO 6 + 1% ZDDP.

comparison of the results showed that the DLC coating in this study had a lower coefficient of friction and wear than that obtained for the CrN and TiN coatings.

XPS analyses were performed in order to understand the chemical interactions of the additives with the Cr-DLC surface. Due to the high content of fluorine in the [BMP][FAP] and [(NEMM)MOE][FAP] additives, this atom was selected to evaluate the interaction between the ILs and the surface. As can be seen in Table 3, the F1s band in the [BMP][FAP] additive may be deconvoluted into two chemical states: a first one at 688.1 eV assignable to interaction fluorine–DLC coating (Gonzalez, et al. (37)) and a second one at 686.3 close to CrF₃ structures from the Cr in the coating (Kemnitz, et al. (42); Kemnitz and Menz (43)). As it is shown in Table 3, samples tested at 20 N showed mainly the DLC-F interacting peak (94%), whereas only 6% was reacting with chromium. FAP anion (FAP⁻) should appear near 688.6 eV (Gonzalez, et al. (37); Blanco, et al. (40)) but it was negligible. When the load was increased to 40 N, the amount of chromium–fluorine interaction was doubled while FAP⁻ remained undetected. This fact is logical taking into account that higher loads will more easily harm the surface.

Cr-F structures (derived from fluorine in the [FAP] anion) in the tribofilm developed by the [(NEMM)MOE][FAP] lubrication mechanism were also detected in previous works with CrN coatings (Blanco, et al. (40)). This could explain the better tribological behavior of DLC and CrN coatings compared to TiN coatings, in which the tribofilm formation appeared to be related to the presence of Ti⁰ and Ti-F structures (Blanco, et al. (41)).

TABLE 3—F1s PHOTOELECTRON BAND OF 1% [BMP][FAP] AND [(NEMM)MOE][FAP] ON THE DLC COATING

Load (N)	[BMP][FAP]		[(NEMM)MOE][FAP]	
	Energy (eV):(%)	Assignment	Energy (eV):(%)	Assignment
20	688.1:94	F coating	688.0:77	F coating
	686.3:6	Cr-F	686.0:11	Cr-F
	688.6:0	FAP ⁻	688.7:11	FAP ⁻
40	688.1:89	F coating	688.1:90	F coating
	686.3:11	Cr-F	686.3:10	Cr-F
	688.6:0	FAP ⁻	688.6:0	FAP ⁻

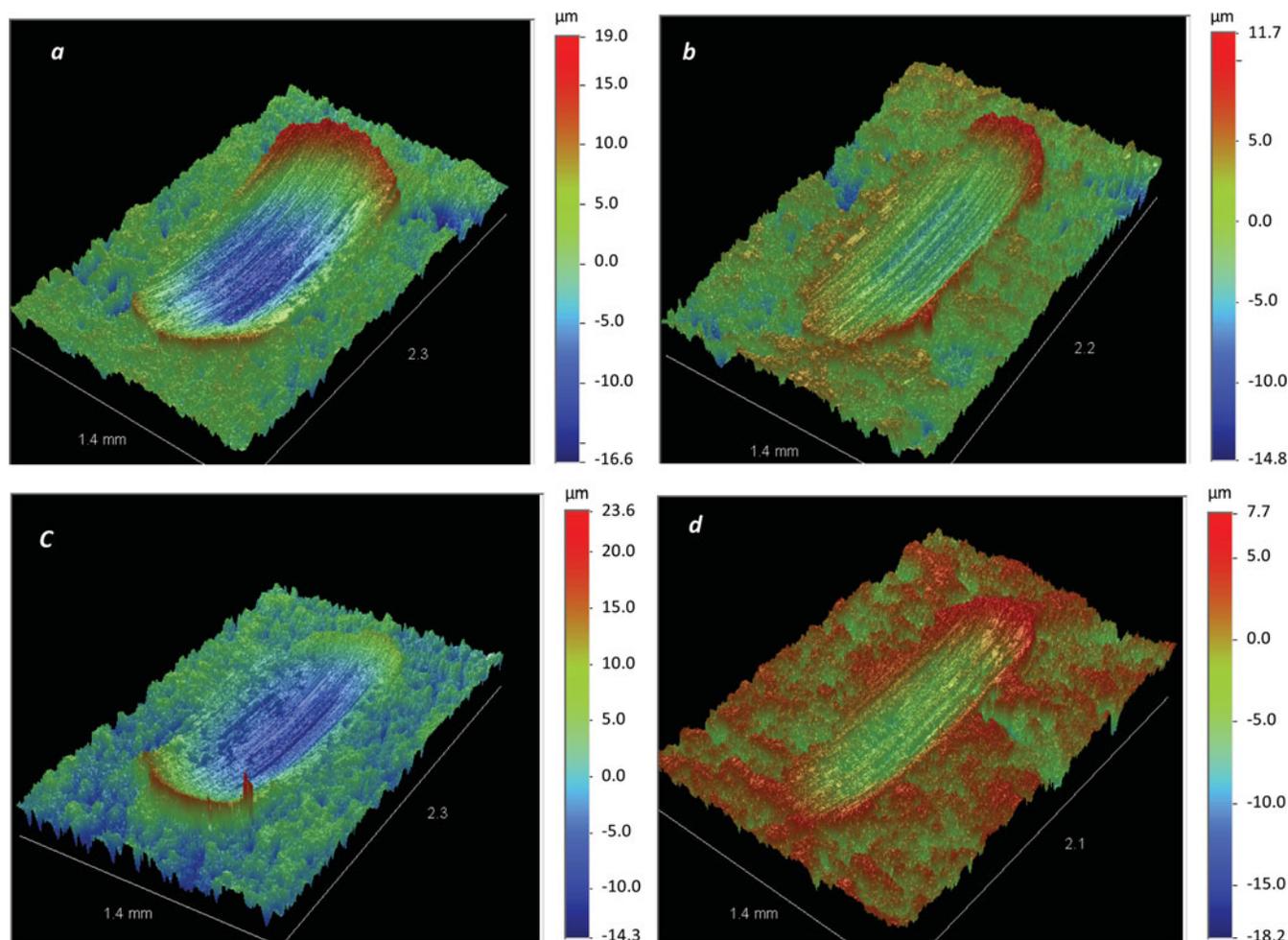


Fig. 5—3D optical wear profiles of the DLC coating after tribological tests at 40 N lubricated with (a) PAO 6, (b) PAO 6 + 1% [BMP][FAP], (c) PAO 6 + 1% [(NEMM)MOE][FAP], and (d) PAO 6 + 1% ZDDP (color figure available online).

It is interesting that the fluorine at 40 N reacted in the same way for both ILs. Tribological results also showed that [BMP][FAP] was slightly better at 20 N than [(NEMM)MOE][FAP]. This difference can be explained by comparing the bands at 688.1 for both lubricants. Additionally, [(NEMM)MOE][FAP]

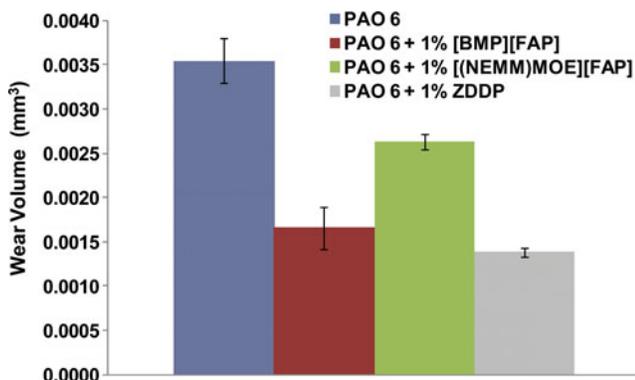


Fig. 6—Average wear volume after tribological tests performed at 40 N (color figure available online).

showed the peak assignable to FAP^- , meaning that to a certain extent the IL did not interact with the surface.

On the other hand, Fig. 7 shows the O1s photoelectron profile of samples lubricated with PAO 6 and all of the mixtures at 40 N. Samples lubricated with neat base oil showed two bands at 530.3 eV (8%) and 532.3 eV (92%) assignable to chromium oxides and organic oxygen, respectively (Blanco, et al. (40); Unnikrishnan, et al. (44); Zhang, et al. (45); Maetaki and Kishi (46)), whereas samples containing 1% of [BMP][FAP] or [(NEMM)MOE][FAP] showed a single O1s photoelectron band at 532.2 or 532.3 eV, corresponding to organic oxygen. A similar behavior was observed when the assay took place at 20 N, with two bands at 530.0 eV (17%) and 532.2 eV (83%) assignable to chromium oxides and organic oxides for the neat base oil sample and a single band at 532.2 or 532.3 eV corresponding to organic oxygen when 1% of [BMP][FAP] or [(NEMM)MOE][FAP] was used.

Furthermore, samples lubricated with ZDDP at 40 N showed a characteristic band of chromium oxides at 529.0 eV (49%) and another at 531.2 eV (51%), which came from phosphates (Haque, et al. (7), (47); Unnikrishnan, et al. (44)). On the contrary, at 20 N a O1s photoelectron band arising from chromium oxides

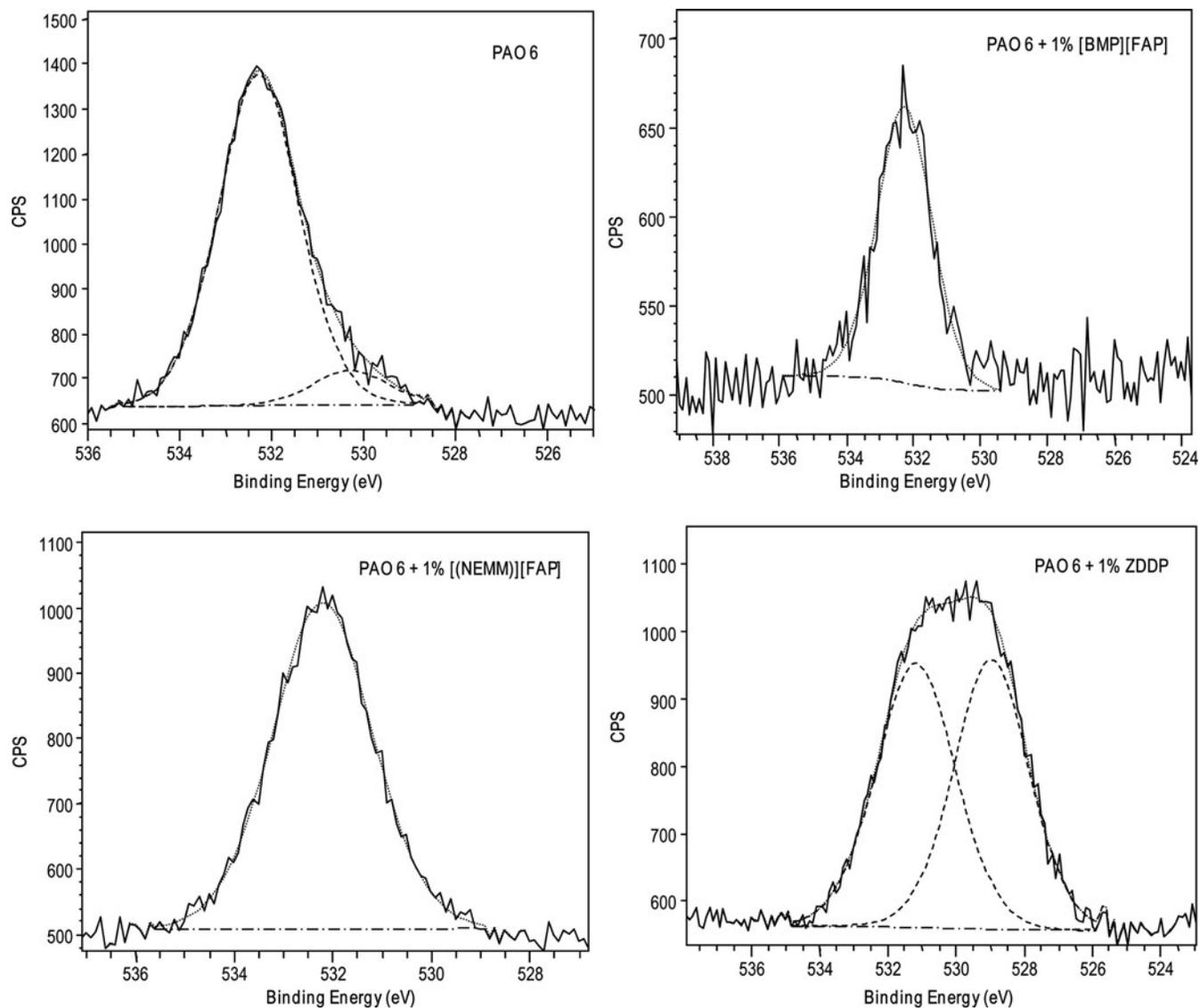


Fig. 7—O1s photoelectron band for samples lubricated with (a) PAO 6, (b) PAO 6 + 1% [BMP][FAP], (c) PAO 6 + 1% [(NEMM)MOE][FAP], and (d) PAO 6 + 1% ZDDP.

disappeared and a new one was present at 532.5 eV (60%), which appeared to come from organic oxygen. The phosphate band was still present at 531.2 eV (40%).

CONCLUSIONS

Ethyl-dimethyl-2-methoxyethylammonium tris(pentafluoroethyl)trifluorophosphate and 1-butyl-1-methylpyrrolidinium tris(pentafluoroethyl)trifluorophosphate ILs as additives to a polyalphaolefin for the lubrication of Cr-DLC coatings were studied and the results were compared with those obtained for traditional ZDDP additive. Tribological tests showed that both ILs reduced friction for the two loads used (20 and 40 N); the mixture with [BMP][FAP] a lower friction coefficient than that with [(NEMM)MOE][FAP] for both loads, but the friction coefficient of the ILs used as additives was slightly higher than that corresponding to ZDDP. An important improvement in the

antiwear performance was also exhibited for all of the additives, resulting in an almost negligible wear track in tests performed at the lowest load. Of the two ILs, at 40 N the mixture with [BMP][FAP] exhibited greater wear reduction, close to the values for ZDDP.

SEM images and XPS analysis indicated that the tribological improvement was due to additive-surface interactions, which promoted the formation of a tribofilm. At 40 N the tribofilm had a different nature depending on the presence of ZDDP or fluorinated ILs. Using the neat base oil (PAO 6), the formation of chromium oxides occurred to a certain extent, ZDDP seems to act in a different way because the presence of chromium oxides was increased and the organic oxygen disappeared in the wear scar. However, the presence of chromium oxides was negligible when fluorinated ILs were present; in this case, the lubrication seems to occur through the formation of chromium fluorides. On the other hand, at 20 N lubrication appeared to take place

through a mechanism similar to that occurring at 40 N with neat PAO 6 or with ILs as additives. However, ZDDP appeared to completely change the mechanism.

ACKNOWLEDGEMENTS

The authors are grateful to the FICYT (the Foundation for the promotion in Asturias of the Applied Scientific Research and Technology) and the Ministry of Science and Innovation, Spain, for supporting this work within the framework of the research projects IB09-047 and DPI2010-18166, respectively. In addition, we would like to thank the Unit for Molecular Spectroscopy and XPS from the Scientific-Technical Services of the University of Oviedo.

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